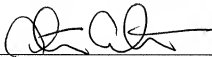


**REMARKS**

The Applicant respectfully requests consideration and allowance of new claims 17 through 39 in view of the amendments presented above. Claims 1 through 16 are canceled above and new claims 17 through 39 are added. These claim amendments are submitted to eliminate multiple dependencies and otherwise place the claims in a U.S. format.

Respectfully submitted,  
THE CULBERTSON GROUP, P.C.

Date: 2/19/08

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1002\_Preliminary Amendment

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CLAMPING AND/OR BRAKING DEVICE

TECHNICAL FIELD OF THE INVENTION

The invention concerns ~~a clamping or braking devices with the features of the preamble of Patent Claim 1~~ a housing a platelike elements held therein to transfer force by bending under pressure.

BACKGROUND

Clamping or braking devices are known in the most varied embodiments for the most varied application cases. Thus, a braking device of a linear guide is known in published European Patent Application No. EP-A-0-936-366 that has a supporting body and can be moved along a guide track. The supporting body has brake shoes, which act on the two longitudinal sides. The supporting body is shaped like an H and has a thin, elastically flexible crosslink and two lower legs, with which it grips the guide track. A brake shoe is located between a lower leg and the guide track. The supporting body is provided with two upper legs, which, together, form a holding space with the crosslink, in which a force-producing means, acting on the upper leg, is provided. This can be hydraulically or pneumatically actuatable toggle lever mechanics or a piezoelectric actuator. Moreover, a hydraulically or pneumatically actuatable wedge valve can be provided as a force transducer, which is conducted in a formed space, which tapers in the longitudinal direction of the brake device, between the upper legs of the supporting body. In all cases, a bending of the elastic crosslink takes place due to the force pressurization of the two upper legs, so that the two lower legs are moved inwards with the brake shoes or apply pressure on the guide track with a large force.

The disadvantage with this known brake device, in connection with the use of an electromechanical transducer such as a piezoelectric element, is, in particular, its sensitivity with respect to vibrations or other mechanical loads.

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1           The use of a toggle lever or a wedge valve, just like an electromechanical transducer,  
2 means a high installation or production expense.

3           In addition, there is, in particular with clamping devices, the requirement of sufficiently  
4 high clamping forces, which, with known devices, can be implemented only with a relatively  
5 high construction and thus financial expense.

6           From U.S. Patent No. 5,855,446, a hydraulic clamping bushing is known, which is  
7 aligned on a shaft and can be connected to it, for example, in a stationary manner. The bushing  
8 essentially has a stable bushing body, which is located at a distance around a drive shaft.  
9 Adjacent to these bushing bodies and facing the shaft, a chamber which can be acted on with  
10 pressure is provided. A side wall of this chamber, which extends essentially parallel to the shaft  
11 is used here, at the same time, as a braking element, which with the action of pressure on the  
12 chamber is pressed against the shaft by the expansion of the chamber, in order to produce a  
13 friction-locking connection. A  $\Sigma$ -shaped formation of the laterally adjacent walls of the chamber  
14 is meant to prevent the bushing from aligning crooked with respect to the wall when the pressure  
15 acts on chamber. The  $\Sigma$ -shaped side walls of the chamber make possible, in the case of pressure  
16 pressurization, an expansion of these side walls in a radial direction toward the shaft, before the  
17 increasing chamber pressure also presses the chamber wall, running parallel to the shaft, against  
18 the shaft. Thus, the bushing can be aligned at right angles to the shaft axis, before the tortional  
19 connection is produced.

20           This clamping device does not have a favorable power transmission for the creation of a  
21 higher pressing force. Furthermore, in its shape and in particular, in its placement possibility  
22 relative to the bushing body, the chamber is limited. The braking effect can be attained here only  
23 by acting on the chamber with excess pressure, and the actual braking element, which must  
24 perhaps transfer high driving forces, is not firmly connected to the bushing body, but rather only  
25 via the inevitably relatively unstable pressure chamber.

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1           From published PCT Patent Application No. WO 01/34990 A1 from the applicant, a  
2 clamping and/or braking device is known, in which a chamber which can be acted on with  
3 pressure is provided on a basic body; in at least one partial area of at least one deformable or  
4 tension-resistant and/or pressure-resistant wall, the chamber is limited. The limitation of the  
5 chamber opposite the wall can also be constructed like the first wall. However, it can also be a  
6 rigid part of a body. Preferably, the walls lie at a relatively small distance from one another. The  
7 forces resulting from the deformation of the chamber are conducted, at least in part, in the  
8 direction of or along the wall and in the area of the connection of the wall to a basic body are  
9 introduced into the latter. If one selects the point of attack of such forces on the basic body, and  
10 if this basic body is, at least partially, elastically deformable, then the forces can be transferred,  
11 for example, into the clamping or braking areas via this basic body on other sites of the basic  
12 body. A corresponding braking or clamping means in these areas can then be moved, by means  
13 of the forces, into a pressurizing position or out of it, so as to brake or release a guide element or  
14 an element which has to be clamped or braked. Both applied excess or reduced pressure in the  
15 chamber can be used thereby, in order to introduce both tractive and compressive forces to the  
16 basic body. Of course, before and after the introduction of force, the clamping and/or braking  
17 areas are still engaged with the guide element or the element to be clamped or braked, wherein,  
18 however, changes in the pressurizing forces between the clamping and/or braking areas and the  
19 other element result.

20           This known clamping and/or braking device is based on the knowledge that when acted  
21 on by a reduced or excess pressure, a suitable chamber tends to a deformation. If this chamber, to  
22 a great extent, is formed by at least one approximately plane wall, then the excess or reduced  
23 pressure in the chamber first causes a deformation in a first direction running essentially  
24 perpendicular to this wall. In order to yield to the deformation (expansion or contraction) in this  
25 first direction, the result from this, in a correspondingly reverse manner, is a contraction or  
26 expansion of the chamber in a second direction generally running perpendicular to the first

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1 direction (that is, essentially parallel to the wall). The fact is used thereby that low forces or  
2 deformations in the first direction produce large forces in the second direction, which can be  
3 used to brake or clamp or to release pretensioned clamping or braking devices.

**SUMMARY OF THE INVENTION**

4  
5  
6 ~~On the basis of this state of the art, the goal of the invention is to create~~ Provided is a  
7 clamping and/or braking device, which can be produced with lower construction expense and  
8 with which sufficiently high braking or clamping forces can be produced in a simple manner.

9 ~~The invention attains this goal with the features of Patent Claim 1.~~

10 The invention is based on the knowledge that by means of at least one plate-like element,  
11 which is located in a housing and has a convex bending area, a clamping and/or braking device  
12 can be produced at low cost; this device can also produce very high clamping and/or braking  
13 forces by means of an only gaseous pressure medium--that is, pneumatically. However, it is clear  
14 that a hydraulic actuation of the device is also possible. The function of the device is guaranteed  
15 by the convex formation of at least one bending area of the plate-like element, with application  
16 on its convex side of an excess pressure from a pressure medium, wherein a corresponding  
17 pressure space is provided on the side of the bending area, which curves convexly outwards.  
18 With pressure acting on the bending area, its curvature is reduced, so that the expansion of the  
19 plate-like element between two ends is increased, of which one is supported on the housing and  
20 the other acts, directly or indirectly, via a pressurizable element, on the object to be clamped or  
21 braked. The plate-like element is thereby so pressure-resistant and constructed so elastically  
22 deformable in the at least one bending area that between the two ends, an elastic element is  
23 formed. The pressure resistance of the plate-like element must be designed in such a way that  
24 when a predetermined, maximum permissible force is exerted on the second end (pressurizing  
25 end), a sharp bending does not occur.

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1           Of course, a corresponding free space, which makes possible a sufficient bending of the  
2 bending area, must be present on the concave side of the bending area, turned away from the  
3 pressure application side. This free space can be connected to the surroundings--that is, to the air  
4 pressure of the surroundings. However, it can also be closed airtight so that the bending of the  
5 bending area must take place, against an increasing pressure, in the free space.

6           Contrary to the known device of the applicant, it is possible, with these embodiments, to  
7 produce or to change clamping and/or braking forces, without hereby having to deform an  
8 essentially rigid body. In this way, aside from a simple construction, a small model results. The  
9 device is, moreover, simple to produce, since only the plate-like element must be placed in the  
10 housing.

11           Contrary to the known clamping bushings, which have relatively soft walls, and which  
12 increase the distance between a pressurizable element and a base element when acted on with  
13 excess pressure, it is also possible with the device in accordance with the invention to produce a  
14 predetermined clamping and/or braking force, with which the object to be clamped or braked is  
15 acted on, even in a pressure-less state during a corresponding pretensioned installation of the  
16 device in the complete device. In this way, for example, a control of the device is possible, in  
17 such a way that a change between a braked (in the pressure-less state) movement of the  
18 pressurized object and a complete clamping of the object (with a sufficient pressurization)  
19 results.

20           The more frequent use of the device in accordance with the invention, however, is the  
21 one where in the pressure-less starting state, the object is not acted on and with a pressurization  
22 of the device, is braked or clamped.

23           With a linearly functioning clamping and/or braking device, the at least one plate-like  
24 element can have a, for example, rectangular shape, wherein the entire element can be convexly  
25 curved between two edges. The first and the second ends are then formed by those edges or end  
26 areas between which the curvature runs. At least in the area of the bending (if the entire plate-

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1 shaped element is not bent), the side edges must be essentially sealed off with respect to the  
2 housing (a small leakage can be tolerated, under certain circumstances, if, for example,  
3 compressed air is used as the pressure medium and the required pressure can be attained and  
4 maintained in spite of the leakage)

5 According to one development of the invention, a pressurizable element is provided,  
6 which is constructed in one piece with the housing and as a part of the housing, or which is  
7 constructed as a part connected to the housing in a detachable manner, wherein the pressurizable  
8 element is constructed deformably in such a way that with a pressure application of the pressure  
9 space, a movement of at least one section of the pressurizable element takes place in the  
10 direction of the object or the transferable clamping and/or braking forces, produced by the at  
11 least one plate-like element, are transferred to the object. With such a deformable pressurizable  
12 element, the pressure space can be sealed off in a simple manner, with respect to the at least one  
13 plate-like element--namely on the pressurizing edge or the second end of the plate-like element,  
14 with respect to the pressurizable element. Basically, however, a pressurizable element is not  
15 connected to the housing can also be used, which, for example, can be constructed  
16 correspondingly large for the distribution of the pressing force on a greater surface. Moreover,  
17 the pressurizable element can be used to adapt the pressing surface (constructed then on the  
18 pressurizable element) to the shape of the object.

19 In a development of the invention, the first end of the at least one plate-like element can  
20 be connected to the housing, especially if in the realization of a linearly acting clamping and/or  
21 braking device, a pressurizable element not connected to the housing is used or the object is  
22 acted on directly by the second end of the at least one plate-like element.

23 In the preferred embodiment of the invention, the first end of the at least one plate-like  
24 part, however, is supported, without a firm connection, against the housing. With the presence of  
25 a pressurizable element, the second end of the at least one plate-like part is also preferably

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1 supported, without a firm connection, against the pressurizable element. In this way, a simple  
2 installation of the device results.

3 According to one embodiment of the invention, the at least one-plate-shaped element is  
4 annular, preferably, circular, so as to create a device that makes possible the clamping of bar- or  
5 rod-shaped objects, in particular, in the cross section of circular-cylindrical objects.

6 The ring-shaped, plate-like element can be shaped convexly, in the starting state, via  
7 essentially the entire radial cross section, wherein essentially the entire ring-shaped wall of the  
8 ring-shaped, plate-like element serves as the bending area. Such a plate-like element can be  
9 produced very simply and at low cost and can be used also for a large number of variants.

10 The at least one ring-shaped, plate-like element preferably has radial slits that are open  
11 inwardly or outwardly, wherein, preferably, the second end--that is, the pressurizing end of the  
12 ring-shaped, plate-like element--is formed by the inside or outside end of the ring-shaped, plate-  
13 like element. The slits preferably extend to just before the opposite end of the ring and are  
14 present in such a number that proceeding from the closed ring area, individual sections extend  
15 inwardly or outwardly. These sections reduce the required deformation of the ring in the azimuth  
16 direction, if the curvature of the convex bending area is reduced during pressurization.

17 With such slit, ring-shaped, plate-like elements, it is possible to provide one or more  
18 sealing elements, preferably, in the shape of a deformable layer, on a part or on the entire surface  
19 of the plate-like element, on the inside and/or outside surfaces of the plate-like elements, relative  
20 to the pressure space, at least in the area of the slits. In this way, a common pressure space can be  
21 used for the individual sections between the slits. The sealing of the slits can, for example, take  
22 place by vulcanizing on a relatively hard, but nevertheless, sufficiently deformable rubber layer.  
23 Such a layer can be produced even on the side of the ring-shaped, plate-like element, turned  
24 away from the pressurization, without the danger of loosening the layer during a pressurization.

25 In the preferred embodiment of the invention, the bending of the at least one bending area  
26 of the at least one plate-like element is limited in such a way that after the discontinuation of the



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1 pressurizing of the pressure space with excess pressure as a result of the elastic effect of the at  
2 least one bending area, it returns to the starting state. One designates, as the starting state, the  
3 pressure-less state, wherein in this state also, depending on the dimensioning or installation of  
4 the clamping and/or braking device, the at least one bending area can be pretensioned. A  
5 corresponding stop for the bending area can be formed, for example, by an inside housing wall.

6 According to one development of the invention, several plate-like elements, arranged  
7 essentially parallel to one another, can be provided, whose second ends have a predetermined  
8 distance to one another, in order to transfer clamping and/or braking forces to the object, via a  
9 prespecified expanded area, wherein for each plate-like element, a separate pressure space or for  
10 several or all plate-like elements, a common pressure space can be formed.

11 When using several plate-like elements, at least one pair of plate-like elements can be  
12 provided, whose first and second ends are directly adjacent or at a close distance to one another,  
13 and whose convex bending areas run convexly outwards, relative to the other plate-like element  
14 of the pair of plate-like elements, wherein a common pressure space is provided for the  
15 pressurization of the bending areas of the two plate-like elements of a pair. This variant offers  
16 the advantage that the two plate-like elements are pressed against one another during a  
17 pressurization and so the space required between the two elements can be sealed off with lower  
18 pressure in a simple manner. For example, the rubber coating mentioned in the preceding can be  
19 provided inside or can be a sealing element connected to the inside surfaces of the plate-like  
20 elements.

21 In addition, the plate-like elements of a pair can be designed in such a way that they lie  
22 close to one another, with at least one partial area of the sides facing one another in an ending  
23 state when acted on with an excess pressure greater or equal to a prespecified maximum  
24 pressure, wherein the ending state is defined by a suitable formation of the bending areas in such  
25 a way that from the ending state, an automatic return to the pressure-less starting state takes

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1 place. In this way, a plate-like element is used as a stop for the at least one bending area of the  
2 other plate-like element.

3 According to one embodiment of the invention, a pressurizable element can be provided,  
4 which is constructed rigid in the entire area in which the several plate-like elements are active. In  
5 this way, a more uniform transfer of the clamping and/or braking forces can be attained on the  
6 object.

7 With all embodiments of the invention, the creation of the clamping and/or braking  
8 forces, however, takes place exclusively or at least, for the most part, predominantly due to one  
9 pressurization and the bending hereby produced and thus the change of the effective expansion  
10 of the plate-like elements between the two ends. This is also true for the case that one surface of  
11 the pressurizable element is acted on completely or partially with pressure, if the bending area(s)  
12 of the plate-like elements are acted on with pressure. The forces hereby produced, which are  
13 comparable with the forces arising with common clamping bushings, can be practically  
14 neglected however.

15 In another embodiment of the invention, the housing and the pressurizable element can be  
16 designed as essentially H-shaped bodies, wherein the at least one plate-like element and the  
17 pressure space are provided between the essentially parallel, upper legs of the H-shaped body,  
18 wherein clamping and/or braking areas are provided on the essentially parallel, lower legs of the  
19 H-shaped body for the transfer of the clamping and/or braking forces to the object and wherein  
20 the H-shaped body is elastically deformable in the area of the middle crosslink and/or the  
21 connecting areas of the middle crosslink with the legs of the H.

22 In a similar manner, the housing and the pressurizable element can be designed as  
23 essentially U-shaped bodies, wherein the at least one plate-like element and the pressure space  
24 are provided between the essentially parallel legs of the U-shaped body, wherein clamping  
25 and/or braking areas are provided on the essentially parallel legs of the H-shaped body for the

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1 transfer of the clamping and/or braking forces on the object and wherein the U-shaped body is  
2 elastically deformable in the area of the base of the U and/or in the area of the legs of the U.

3 Other embodiments of the invention ~~can be deduced from the subclaims~~ may be described  
4 or claimed herein.

5 The invention is explained in more detail below, with the aid of the embodiment  
6 examples shown in the drawing. The figures in the drawing show the following:

BRIEF DESCRIPTION OF THE DRAWINGS

7  
8  
9 Figure 1a ~~is~~ a longitudinal section (~~Figure 1a~~) and a detail enlargement (Figure 1b) ~~is a~~  
10 detail enlargement of a first embodiment of a clamping and/or braking device, according to the  
11 invention, for the pressurizing of a circular-cylindrical object, using a ring-shaped, plate-like  
12 element<sub>1</sub>.

13 Figure 2<sub>1</sub> ~~is~~ a top view of the ring-shaped, plate-like element in Figure 1 and a detail  
14 enlargement in sectional view along the line A-A (Figure 2b)<sub>1</sub>.

15 Figure 3a<sub>1</sub> ~~is~~ a longitudinal section (~~Figure 3a~~) and Figure 3b ~~is a~~ detail enlargement  
16 (~~Figure 3b~~) of a second embodiment of a clamping and/or braking device according to the  
17 invention for the pressurization of a circular-cylindrical object, using several pairs of ring-  
18 shaped, plate-like elements<sub>2</sub>.

19 Figure 4<sub>1</sub> ~~is~~ a detail enlargement of a third embodiment of a clamping and/or braking  
20 device according to the invention, analogous Figure 3, but using a ring-shaped pressurizable  
21 element<sub>3</sub>.

22 Figure 5<sub>1</sub> ~~is~~ a detail enlargement of a fourth embodiment of a clamping and/or braking  
23 device according to the invention, analogous Figure 4, but using a pressurizable element,  
24 constructed as one piece with the housing<sub>3</sub> and

25 Figure 6<sub>1</sub> ~~is~~ a schematic cross section through a fifth embodiment of a clamping and/or  
26 braking device according to the invention for the realization of a linear guide.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

Figure 1a shows a schematic longitudinal section of an and/or braking device 1, shown in Figure 1 in a schematic longitudinal section, has with a ring-shaped housing 3, which consists of a first part 5 with a ring-shaped recess 7 and a second part 9, which is connected to the first part 5, for example, screwed to it. In the recess 7, between the first and second housing part 5, 9, a ring-shaped, plate-like element 11 is located. The ring-shaped, plate-like element 11 is shaped slightly convex in the radial section and applies pressure on the circumferential wall of the ring-shaped recess 7 of the first housing part 5 with its outside end or its outside boundary 11a.

As can be seen from Figure 1b, the ring-shaped, plate-like element 11, consisting, for example, of a steel sheet, can be connected, on one side, to a sealing element 13, for example, a relatively hard, but still sufficiently deformable rubber ring. The ring-shaped, plate-like element 11 is, as shown in Figures 2a and 2b, provided with a large number of slits 14, which proceeding from an outside boundary area 11c, which is also closed in the shape of a ring, run radially inwards and open toward the inside.

The ring-shaped, plate-like element 11 defines, together with the sealing element 13, on its convex side, together with the inside wall of the first housing part 7, a pressure space 15, to which a pressure medium can be supplied, for example, compressed air. Of course, however, a liquid pressure medium can also be used. The pressure space 15 is constructed open toward the inside in the embodiment shown in Figure 1 and must be sealed off, with respect to the object 17 to be acted on, for example, a rotatable shaft. This can take place by means of corresponding sealing elements 19, which can be designed as O rings. The O rings 19 can be taken up in grooves in the inside walls of the first and second housing part 5, 9.

The ring-shaped, plate-like element 11 lies close to the inside wall of the second housing part 9 in the embodiment example shown in Figure 1, with its outside and inside boundaries 11a, 11b, wherein the sealing element sufficiently seals off the space between the concave surface of

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1 the element 11 and the inside wall of the housing part 9. In this respect, an annular area 13a, 13b  
2 of the sealing element can be constructed on the outside and inside boundary. The inside annular  
3 area 13b can be designed so far, in its radial expansion, that it lies close to the inside wall of the  
4 housing part 9, at least with a partial area over the entire movement path of the element 11. The  
5 space between the concave side of the element 11 and the housing part 9 can also be connected  
6 to the surroundings, via one or more (not depicted) canals.

7 If the pressure medium is conducted under a certain pressure to the pressure space 15, via  
8 one or more (not depicted) supply canals, then the ring-shaped, plate-like element 11 is bent in  
9 the direction of the second housing part 9--that is, the curvature is reduced. In this way, the  
10 sections of the element 11, stretching inwards between the slits 14, are moved, with its inside  
11 boundary 11b, further inwards in the direction of the object 17, so that a desired braking or  
12 clamping force can be exerted on the object by a certain pressure of the pressure medium. The  
13 element 11 must, of course, be so dimensioned and be made of such a material that at least in one  
14 bending area, there is a sufficient deformability and moreover, the element 11 is constructed  
15 sufficiently pressure-resistant as a whole that even with the maximum producible clamping or  
16 braking force, which acts on the inside end of the element 11 or is exerted on the object 17 by it,  
17 no sharp bending of the element 11 or even a plastic deformation occur.

18 The inside wall of the second housing part 9 is also used as a stop for the ring-shaped,  
19 plate-like element 11, so that even with high pressures, an abrupt reverting of the convex  
20 curvature to a concave curvature cannot occur. Moreover, the inside wall of the second housing  
21 part 9 and/or the ring-shaped, plate-like element 1 or the sealing element 13 can be constructed  
22 in such a way that the curvature of the element 11 is reduced, as a maximum, so much that with a  
23 subsiding of the pressure, the elastic effect of the element 11 ensure that the element 11 again  
24 returns to the curvature present in the pressure-less starting state. In this respect, the inside wall  
25 of the housing part 9 can be constructed, for example, slightly convex in the direction of the

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1 element 11 or one or more stop elements can be provided on the inside wall of the housing part 9  
2 or on the element 1 or on the sealing element 13.

3 At this point, it should be noted that the sealing element can, of course, also be provided  
4 on the outside convex side of the element 11. In this case, it is also possible to use a loose sealing  
5 element, not connected to the element 11, for example, a separate, ring-shaped sealing disk made  
6 of rubber or plastic.

7 The embodiment of a clamping and/or braking device shown in Figure 3 is similar to the  
8 embodiment in Figure 1. Instead of a single ring-shaped, plate-like element 11, five pairs of ring-  
9 shaped, plate-like elements 11 ~~erroneously labeled as 7 in Figure 3a~~, as a whole, are, however,  
10 provided in this embodiment. In this way, ten times the braking or clamping forces can be  
11 transferred to the object 17 with the same pressure of the pressure medium.

12 The two ring-shaped, plate-like elements 11 of a pair are placed thereby in such a way  
13 that their concave sides oppose each other. In this way, a space sealed-off with respect to the  
14 pressure space 15, in which the elements 11 of the pairs of elements, sealed off with respect to  
15 each other, is formed, in a simple manner, between each pair of elements 11. With an  
16 pressurization of the pressure space 15 with the pressure of the pressure medium, the slit walls of  
17 the elements 11 are there to be bent toward one another. Since the inside ends 11b of the  
18 elements 11 of a pair move jointly, a sealing off of the space between the elements 11 can be  
19 reliably attained with simple means. The outside boundaries of the elements 11 can, of course,  
20 also be held, for the fixing, by suitable means in the housing 3, for example, in grooves in the  
21 circumferential wall of the cylindrical recess 7 of the first housing part 5. The two elements 11 of  
22 a pair can, of course, also be connected to one another—for example, they can be locked together.

23 The embodiment of a clamping and/or braking device 1, according to Figure 4, differs  
24 from the embodiment shown in Figure 3 merely in that the inside ends 11b of the elements 11 do  
25 not apply pressure directly on the object 17 to be braked or to be clamped, but rather, via a ring-  
26 shaped pressurizable element 21, which is located inside the housing 3. The pressurizable

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1 element 21 is so deformed by the forces exerted by the pairs of ring-shaped, plate-like elements  
2 11--in particular, the inside diameter of the pressurizable element 21 is reduced to such an extent  
3 that the forces are transferred, at least partially, to the object 17.

4 This embodiment offers the advantage that the sealing off of the pressure space 15  
5 between the housing and the pressurizable element 21 can take place--for example, again via the  
6 sealing elements 19 in the form of O rings.

7 As can be seen from Figure 5, in which another embodiment of a clamping and/or  
8 braking device is shown, it is also possible to provide a pressurizable element 21, constructed in  
9 an integrated manner with the housing 3, instead of a separate pressurizable element. In the case  
10 shown, the recess 7 in the first housing part 5 is designed in such a manner that a thin inside wall  
11 is formed, which forms the pressurizable element 21. The wall must be so thin that a sufficient  
12 deformation can be guaranteed when forces are exerted by the pairs of ring-shaped, plate-like  
13 elements 11 and thus a sufficient transfer of these forces to the object 17. The sealing off of the  
14 pressure space 15 can take place here in that the thin wall or the pressurizable element 21 is  
15 connected to the second housing part 9 in an essentially airtight manner. For the sealing, a  
16 sealing element 19, for example, in the form of an O ring, can again be used.

17 Of course, such a clamping and/or braking device can also be designed in such a way that  
18 the corresponding forces are transferred to an object on a point-like, linear, or non-ring-shaped  
19 closed surface, for example, for the production of a clamping and/or braking device for a linear  
20 guide.

21 If one does not interpret the representation in Figure 5 as an excerpt from an essentially  
22 rotation-symmetrical representation, but rather as an excerpt from a representation with a (for  
23 example, rectangular) housing 3, in which rectangular, plate-like elements 11 are placed, then a  
24 clamping and/or braking device is formed for the pressurization of the surface of an object 17, in  
25 which, for example, it may be a track of a linear guide. The housing 3 can, for example, be  
26 mounted on a carriage, not depicted in more detail, conducted by means of the object 17

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1 designed as a track. The carriage can grip the track so that the exertion even of high forces on the  
2 surface of the object is possible, without lifting the carriage off the track. The mode of  
3 functioning of, for example, rectangular, plate-like elements can be deduced analogous to the  
4 preceding statements in connection with the annular, plate-like elements.

5 If a pressurizable element 21 is not used with such an embodiment, then the plate-like  
6 elements 1 can be held, with their inside ends 11a, in the housing or connected firmly with it  
7 (one piece or detachable). Of course, a sealing off of the pressure space 15, with respect to the  
8 object 17, must then again take place. The lateral edges or areas of the rectangular, plate-like  
9 elements must also be sealed off to create a pressure space or a "pressure-less" space between the  
10 pairs of plate-like elements.

11 The embodiment according to Figure 6 is also a version of a clamping and/or braking  
12 device 1, with which a linear guide can be realized. The first housing part 5 is constructed, in this  
13 case, with an H-shaped cross section, wherein between the upper legs of the H, a pair of, for  
14 example, rectangular, plate-like elements 11 is provided. Since such elements 11 need not be  
15 provided with slits, it is possible, of course, to also dispense with corresponding sealing  
16 elements. The pressure space 15 is, in turn, formed outside the pair of the plate-like elements 11.

17 In a pressurization of the pressure space 15, a deformation of the H-shaped housing part 5  
18 is produced, in particular, in the area of the middle crosslink of the H and the transition areas  
19 between the crosslink and the lateral legs of the H. In this way, the rigidly constructed, lower  
20 legs of the H are moved inwards in the direction of the intermediately placed object 17 and  
21 transfer corresponding braking or clamping forces to the object. In this respect, it is possible to  
22 provide clamping and/or brake shoes 23 on the insides of the lower legs of the H.

23 If instead of an H-shaped housing part 5, a U-shaped housing is used, in which one or  
24 more (for example, rectangular) plate-like elements, are provided between the legs of the U, with  
25 these elements pressing the legs of the U toward the outside with a pressurization, then an "active  
26 loosening" of an object, for example, with a linear guide, can be realized. Of course, this can also



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1 be attained with an H-shaped housing in that the brake shoes are located on the outsides of the  
2 lower legs of the H and they apply pressure on the object, for example, in a groove. Conversely,  
3 an "active clamping or braking" can also be produced with a U-shaped housing, if the brake  
4 shoes are located on the outsides of the legs of the U and correspondingly apply pressure on the  
5 object.

6 Finally, it should be pointed out that of course, individual features which are described in  
7 the preceding only in connection with a specific embodiment can also be combined, if  
8 reasonable, with other embodiments also. The described embodiments illustrate the scope of the  
9 claims but do not restrict the scope of the claims.

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12 1002\_Exhibit A\_Marked Specification